

PLUG-IN CONNECTOR

The invention relates to a plug-in connector in accordance with the features provided in the preamble to patent claim 1.

Such plug-in connectors are known from DE 199 35 246 A1 and DE 102
5 21 971 A1, and contain a base body, an annular sealing element, a retaining
element, and a sleeve that can be connected to the base body. A tube end that can
be inserted into an annular gap between the base body and the sleeve can be fixed
in the plug-in connector by means of the annular retaining element, which is also
called a clamping ring, whereby the annular sealing element arranged on the base
10 body is sealingly positioned against the interior surface of the inserted tube end.
The retaining element is divided largely in the direction of the longitudinal axis of
the plug-in connector by a gap and contains at least one claw that engages radially
in the exterior surface of the tube end for fixing the latter in the plug-in connector.
There can be difficulties in terms of functional fixation of the tube end in the
15 plug-in connector, in particular as a result of production tolerances for the sleeve
and for the retaining element that is divided in the longitudinal direction and
furthermore for the tubes. Moreover, problems can occur due to the claw or claws
of the retaining element penetrating too deep into the exterior surface of the tube,
whereby disadvantageous damage to the intermediate metal layer can occur when

the tube is embodied as a plastic/metal compound tube with an intermediate metal layer that is in particular made of aluminum.

Starting at this point, the object of the invention is to further develop the plug-in connector of the type cited in the foregoing such that high functionality of the connecting and fixing of the tube end in the plug-in coupling is attained with low production and assembly complexity. The plug-in connector should facilitate functionally correct handling during assembly, and should preclude the risk of assembly errors. The plug-in connector should also be able to be used for tubes made of different materials, such as plastic or metal or a composite thereof in the form of compound tubes, and for tubes made of comparatively soft materials or plastics, such as in particular polyethylene. It should be possible to produce the plug-in connector with no problem, and the plug-in connector should ensure a lasting and secure connection of tube to plug-in connector for many years. Furthermore, the plug-in connector should be able to be used in the field of sanitation, in particular for hot water and cold water installation. Above all, the plug-in connector should be able to reliably withstand changes in the material, in particular with regard to the dimensions or rigidity of the tube, that are caused by changes in temperature in the flowing medium, whereby functional sealing and connection should be assured for a service life that is decades-long.

This object is achieved in accordance with the features cited in patent claim

1.

The inventive plug-in connector is distinguished by a functional design and by low production and assembly complexity. The base body that partially engages in the interior of the tube end ensures secure radial support of the tube end, so that the retaining forces are securely received and transmitted by means of the retaining element and there is secure sealing, even under changing operating conditions.

The retaining element is embodied as a ring that is closed over the circumference.

Furthermore contains an annular part that is closed radially exteriorly over the circumference. One contains the retaining element radially, in particular about a pre-specified angle claws inclined to the longitudinal axis, whereby the closed ring part is fixed in the sleeve and/or is integrated therein. The sleeve that is connected

to the base body preferably contains two sleeve parts that are securely joined to one another, whereby the closed annular part of the retaining element is established in

the connecting area of the two sleeve parts. When in the unloaded condition, the tongue-like claws of the retaining element in which the tube end is not inserted stand at a pre-specified angle to the longitudinal axis of the plug-in connector, whereby the aforesaid angle opens in the direction of the anterior free sleeve end and the edges of the aforesaid retaining claws, which edges are inclined to the

longitudinal axis, point into the interior in the direction of the second posterior sleeve part.

In a preferred manner, allocated to the claws, hereinafter the retaining claws, of the retaining element is a support body situated on their interior side that
5 faces the longitudinal axis. The support body is in particular embodied as an at least approximately conical annular shoulder of the anterior sleeve part. Alternatively, the support body can be embodied from segments, cams, or the like. Precise centering of the retaining element when the two sleeve parts are being joined is assured by means of the support body. Preferably the penetration depth
10 of the retaining claws into the tube end is limited by means of the support body, this being of particular importance especially with compound tubes. The geometry of the sleeve is provided such that while the tube end is being inserted the interior diameter of the retaining element, i.e. its retaining claws, can enlarge by a pre-specified amount, for instance on the order of magnitude of 0.6 mm, so that
15 the retaining claws lie against the exterior of the tube, i.e., on the tube exterior jacket, with pre-stress. When there is tensile strain, that is counter to the direction of insertion, the retaining claws only penetrate outward into the inserted tube by a pre-specified maximum amount. As a result of the inventive support element and/or the conical annular shoulder, the penetration depth is limited to a
20 pre-specified maximum amount, preferably on the order of magnitude of 0.4 mm.

It should be noted at this point that given a compound tube with a dimension of 16 mm and with the tube having a total wall thickness of 2 mm, the exterior plastic layer, in particular the polyethylene layer, is 0.5 mm thick. In the case of greater tube wall thicknesses, in particular for tubes with dimensions of 20, 25, 32, and 40 mm, it is known that the tube wall thickness is correspondingly larger and is for instance 0.6 mm. In accordance with the invention, the support element is embodied such that and/or matched to the tube wall strength and in particular to the exterior layer of a compound tube such that the maximum penetration depth is less than the exterior layer of the tube by a pre-specified amount, so that the retaining claws are reliably prevented from penetrating into the metal layer.

The two sleeve parts, which are in particular embodied as plastic injection-molded parts, are preferably securely and lastingly joined to one another by means of ultrasonic welding. Prior to performing the welding, the retaining element is arranged between the two sleeve elements, whereby the retaining element is precisely centered by means of the aforesaid support element, in particular embodied as a conical annular shoulder, of the anterior sleeve part. The posterior sleeve part has a pre-determined welding geometry such that during the welding process its material is melted and used for additional weld material. Once joined, in particular once joined by welding, the sleeve is a uniform compact

component into which the retaining element is preferably non-detachably integrated. A material-fit connection between the two sleeve parts is realized in a preferred manner using the welding, and the finished sleeve is a uniform, ultimately single-piece component. The retaining element is prevented from
5 detaching from the sleeve, and incorrect positioning of the retaining element in the sleeve, and resultant malfunctions, especially reductions in the retaining and withdrawing forces, are also avoided. Alternatively, the sleeve can be produced and/or embodied in one piece as a plastic injection-molded part, whereby the retaining element is integrated directly into the sleeve during production.

10 As described in the foregoing, the retaining element contains an annular part that is fixed between the sleeve parts and/or integrated into the sleeve after the welding, so that, in contrast to an open retaining element or a retaining element embodied with an axial gap, there are no disadvantageous tolerance chains with the associated receiving diameters of the sleeve, whereby furthermore substantially
15 improved dimensional stability is attained for the retaining element. The insertion forces and/or retaining forces are optimized due to the arrangement of the retaining claws at a pre-specified angle to the longitudinal axis, preferably between 20 - 45°, in particular between 25 - 40°, and usefully between 28 - 35°, and/or due to the exterior placement surface, preferably embodied as a cone, of the support body or
20 annular shoulder.

The retaining claws are also very flexible as a result of the thus attained relative extension of the retaining claws, with the simultaneously relatively small receiving diameter of the retaining element in the sleeve and/or a small exterior diameter of the sleeve. The tube end can thus be inserted with a substantially
5 minimized expenditure of force. Furthermore, the claws have a special shape in that the edges of the retaining claws that engage the exterior surface of the tube are embodied rounded and/or rounded off. This attains an additional reduction in the insertion force, since when it is inserted the tube end first strikes the exterior edges of the retaining claws. In addition, in the case of tensile force out of the plug-in
10 coupling, the fact that the claw edge progressively penetrates into the tube effects an increase in the retaining force. The retaining claws inventively have an arched claw shape and/or scoop shape, which provides good dimensional stability.

Particular embodiments and further developments of the invention are provided in the subordinate claims and in the following description.

15 The invention is described in greater detail in the following using the exemplary embodiments in the drawings without this constituting a restriction.

Fig. 1 is an axial section through the plug-in coupling;

Figs. 2, 3 are axial sections of the two sleeve parts and of the welded sleeve created thereby, including retaining element;

Fig. 4 provides a partial illustration and enlargement of the plug-in coupling with inserted tube end;

Fig. 5 is a perspective drawing of the retaining element;

Fig. 6 is an axial section of an alternative exemplary embodiment
5 of the sleeve;

Fig. 7 is an axial section through the base body;

Fig. 8 is one exemplary embodiment of the plug-in coupling in the shape of a T-piece;

Fig. 9 is another exemplary embodiment of the plug-in coupling as
10 a wall disk;

Fig. 1 is an axial section of the plug-in coupling, which contains a base body 2, a sleeve 4 joined thereto, an annular retaining element 6, and an annular sealing element 8. The sleeve encloses a portion of the base body 2, forming an annular gap 10 into which one end of a tube can be inserted, whereby the retaining
15 element 6 with retaining claws 12 engages the exterior surface of the tube end with pre-stress and the sealing element 8 arranged on the exterior of the base body 2 is sealingly positioned against the interior surface of the tube end. The anterior free sleeve end 14 of the sleeve 4 projects for a pre-specified distance 16 beyond the free end 18 such that when the tube end is inserted into the annular gap 10 the tube
20 end is securely guided and centered in terms of the plug-in coupling and there the

sealing element 8 is prevented from being damaged, squeezed off, or even from being pressed away from the base body 2. The sleeve 4 is securely joined to the base body 2, usefully via locking means 20, 22 that correspond to and/or engage in one another. As shown, the one locking means 20 of the base body 2 is embodied as a radial bar that engages in the other locking means 22, which is embodied as an annular groove of the sleeve 4. Alternatively, the base body 2 can have an annular groove for a bar of the sleeve 4 that is inwardly oriented. Although a lasting and/or non-detachable connection having locking means has proved particularly useful, in the framework of the invention the connection between the base body 2 and the sleeve 4 can also be embodied to be detachable, whereby in this case a screwed connection or a bayonet connection is possible. It should furthermore be noted that the plug-in coupling can be a component of a fitting or can be embodied as such and likewise can be integrated into a fitting or other device such that the base body is embodied in a single part, for instance with the housing of the fitting.

In the direction of the longitudinal axis 24, the sealing element 8 is arranged at a distance from the area in which the retaining element 6 is provided. Due to the axial space, the retaining element 6 and the sealing element 8 can be dimensioned independently of one another, there being no need to worry about mutual interference in the different functions. The radially exteriorly arranged,

preferably cylindrical interior surface 26 of the anterior part of the sleeve 4 thus facilitates precise and/or secure tube support in the area of the sealing element 8. Impermissible broadening of the tube end due to pressure and/or mechanical stresses in the area of the sealing element are prevented, and the required pressure by the sealing element against the interior surface of the tube end is maintained.

The posterior sleeve part 32 also contains an interior surface 28 that faces the inserted tube end. In a preferred manner, the interior diameter of this interior surface 28 is at least approximately the same size as that of the exterior diameter of the tube end to be inserted, in particular as the maximum exterior diameter as pre-specified by tolerances. This results in optimum guidance and stabilization of the tube end. In contrast, the interior surface 26 of the anterior sleeve part 30 has an interior diameter that is larger than the interior diameter of the posterior sleeve part 32 in the area of its interior surface 28 by a pre-specified amount, in particular on the order of magnitude of up to several tenths of a millimeter. This reduces the inserting force to a minimum, on the one hand, and ensures optimum guidance and stabilization of the inserted end of the tube, which is preferably embodied as a compound tube.

Fig. 2 depicts the first anterior sleeve part 30 and the second posterior sleeve part 32 prior to their being joined or welded. Fig. 3 depicts the finished sleeve 4 with the integrated retaining element 6 after the joint, in particular a weld

joint, has been produced. The first sleeve part 30 contains a support body 34 for the retaining element 6 and/or its retaining claws 12, the free edges of which 38 are indicated only schematically here and are on a smaller radius than the radius of the interior surface 26 of the first, anterior sleeve part 30. When the tube end is inserted, the claw edges 38 are pressed radially outward and consequently, under clamping or pre-stress, engage the exterior surface of the tube end for the purpose of fixing the latter in the plug-in coupling. The support body 34 is preferably embodied as a conical annular shoulder, in particular of the first anterior sleeve part 30, and/or is an integral component of the first sleeve part 30 and thus after the two housing parts 30, 32 have been combined is an integral component of the complete sleeve 4. While the two sleeve parts 30 32 are being joined, the retaining element 6 is centered with respect to the longitudinal axis 24 in a particularly useful manner by means of the support body 34, in particular the annular shoulder having a conical exterior surface. As can be seen, the first sleeve part 30 and the second sleeve part 32 have connecting parts 40, 42 that are matched to one another and/or that engage one another and that enable precise mutual orientation as well as simple and functional, in particular lasting, joining, preferably a weld joint, of the two sleeve parts 30, 32.

For the precise placement of the retaining element 6 between the two sleeve parts 30, 32, the retaining element 6 has radially outwardly an annular part 44 that

is closed on the circumference and that is preferably arranged largely in one radial plane that is orthogonal to the longitudinal axis 24. As can be seen in particular in Fig. 2, the closed annular part 44, which thus does not have a slit that runs in the axial direction, is embodied curved somewhat spherically on the radial plane in the manner of a toroid. When the sleeve is assembled, this usefully results in a tolerance compensation, whereby the retaining element is also centered, in particular via the retaining claws 12 by means of the support body 34 and . Once joined, in particular once the material-fit weld joint has been produced, in accordance with Fig. 3 the sleeve 4 is a single-part and/or compact component in which the retaining element is integrated and/or precisely secured. The retaining element 6 is also fixed both radially and axially with the closed annular part 44 in the sleeve 4, whereby the resiliently elastic retaining claws 12 are pressed outward in the direction of the arrow 46 when a tube end is inserted and thereafter act with pre-stress to securely clamp the exterior surface of the inserted tube end with a pre-specified clamping tension. The closed annular part 44 ensures that the retaining element 6 is immovably fixed, both axially and radially.

As can furthermore be seen from Fig. 3, the sleeve 4 and/or the second sleeve part 32 in the area of the retaining claws 12 has an interior surface part 48 that preferably expands conically in the direction of the free sleeve end 14 such that the resiliently elastic retaining claws 12 can pass through the radial expansion in

the direction of the arrow 46. It must be expressly stated at this point that the retaining element 6 comprises a resiliently elastic material, in particular spring steel or the like. The support body or conical annular shoulder 34 has an exterior positioning surface 50 for the retaining element or its retaining claws 12. The positioning surface 50 is preferably embodied largely conical and is situated at an angle 52 to the longitudinal axis 24, whereby when not stressed the retaining claws 12 are disposed at largely the same angle to the longitudinal axis 24. The angle 52 is pre-specified in the range of 20° to 45° , usefully 25° to 40° , preferably 28° to 35° , in particular 30 to 33° . The interior surface part 48 is arranged at an angle 54 such that when a tube end is inserted the retaining claws 12 can be spread unimpeded into the free space 56 that is limited radially outward by the interior surface part 48. The angle 54 is advantageously smaller than the angle 52 by a pre-specified factor, whereby the factor is in the range of 0.3 to 0.7, advantageously 0.4 to 0.6, and in particular approximately 0.5. It should also be mentioned that the sleeve 4 and/or the posterior sleeve part 32 contains at least one, usefully a plurality of, radial through-holes or viewing windows 58 through which it is possible to check that the tube end has been correctly and completely inserted into the plug-in coupling.

Fig. 4 is a partial and enlarged depiction of the plug-in coupling with a tube, the end 60 of which has been inserted into the annular gap between the base

body 2 and the sleeve 4. The two sealing bulges of the sealing element 8 that are arranged radially exteriorly are positioned against the interior surface of the tube end 60, while the retaining claws 12 of the retaining element 6 engage the exterior surface of the tube end 60. The tube end 60 is embodied as a compound tube with an intermediate metal layer 62. Due to the inventive support body 34, which is preferably embodied as a conical annular shoulder, and in combination with the design of the length of the retaining claws, it is assured that the retaining claws 12 can be securely supported and cannot penetrate to the metal intermediate layer 62, damaging it, in particular in the case of tensile forces in the direction of the arrow 64 on the tube end 60.

Fig. 5 provides a view of the retaining element at a viewing angle that is inclined to the longitudinal axis. The retaining element 6 contains the annular part 44 closed over the circumference as well as eight scoop-shaped and/or arched retaining claws 12. The annular part 44 is arranged in the sleeve in a plane that is largely orthogonal to the longitudinal axis 24, which here is inclined to the plane of the drawing. As explained in the foregoing, the scoop-shaped retaining claws 12 are arranged inclined to the longitudinal axis 24 at a pre-specified angle, whereby there is an angling-off 66 between the annular part 44 and each of the retaining claws 12. As can be seen in accordance with the retaining claws at the bottom of the drawing, the retaining claws 12 are arched in a pre-specified radius

that is matched to the radius of the exterior surface of the tube end. Between the retaining claws 12, the retaining element 6 contains recesses 68, the corners of which are usefully rounded and which open in the direction of the claws edges 38 into open through-holes 72. The retaining claws 12 thus contain expansions 72 that extend to both sides in the circumferential direction relative to the free claw edges 20. At the closed annular part 44 and between the recesses arranged spaced apart in the circumferential direction, the retaining claws 12 embodied as resiliently elastic tongues then have a metal part 74 to which the aforesaid expansions 72 are connected.

Because of the predetermined arch or curvature, relative to the longitudinal axis 24, of the retaining claws 12 embodied in a scoop shape, the exterior edges 76 of the expansions 72 have a radius 78 to the longitudinal axis 24 that is smaller by a pre-determined amount than the radius 80 in the area of the center of the claw edges 38. Given the inventive in particular claw-shape and/or the rounding due to the claw edge 38 that engages the tube end, when inserted into the plug-in coupling the tube end first strikes the exterior edges 76 of the retaining claws 12. Due to the special shape of the retaining claws, which has been described, the insertion force for the tube end is advantageously reduced, specifically preferably in combination with the comparatively long design and the lever arm of the resiliently elastic retaining element 12 that is arranged inclined to the longitudinal axis.

Furthermore, due to the design of the retaining element, explained in the foregoing, tensile forces on the tube end in the direction out of the plug-in coupling cause the claw edge 38 to progressively penetrate into the tube and increase the retaining force. In addition, it is explicitly noted that due to the arched design of the retaining claws 12, the latter have high dimensional stability. Since the retaining element 6 is immovably fixed in the sleeve axially and radially by means of the closed annular part 44, when forces act thereupon undesired movements of the retaining element 6 relative to the sleeve and/or the tube end are prevented, and thus resulting local overstressing of the retaining claws 12 is prevented and they are prevented from penetrating too deep into the tube end.

Fig. 6 illustrates an alternative embodiment of the sleeve 4, the sleeve parts 30, 32 of which are joined by means of a clip or snap-in connection. The sleeve parts 30 and 32 have locking elements 82, 84 that correspond to and engage one another, whereby in accordance with the explanations in the foregoing the retaining element 6 is positioned securely between the two sleeve parts 30, 32. Otherwise the statements in the foregoing still apply.

Fig. 7 is an axial section of a special embodiment of the base body 2 with the sealing element 8 arranged in an annular groove 86. The annular groove 86 contains an annular body 88 that is outwardly oriented and that engages in a corresponding and radially inwardly positioned recess 90 of the sealing element 8.

In a preferred manner, there is an adhesive connection 94 between the anterior section 92 of the sealing element 8 and the annular groove 86. In contrast, the posterior section 96 of the sealing element 8, which follows in the direction of the interior of the plug-in coupling or base body 2, is arranged free and/or loose in the annular groove 86. Due to the adhesive connection 94 of the sealing element 8, which is present only in the area of the anterior section 92, the sealing element 8 is held securely in place while the tube end is inserted. On the other hand, there is no increase in the insertion forces, because the posterior section 96 of the sealing element 8 is flexible and/or movable in the axial direction. The sealing element 8 is provided with slide means, at least radially on the exterior, specifically for instance it is provided during assembly with fat or the like or is coated with an appropriate lubricant during manufacture. It should be noted that forces during insertion of the tube end are kept low due to the lubricant, and furthermore damage to or even destruction of the sealing element 8 is prevented.

Figure 8 provides a partial side view and partial cut-away view of a special embodiment of the inventive plug-in coupling in the form of a T-part for connecting three tube ends. The three coupling parts are embodied to match, whereby the two coaxial coupling parts have the sleeves 4, while the left-hand coupling part is illustrated without the sleeve for the purpose of clarifying the structural embodiment of the base body 2. The T-shaped base body 2 and the

sleeves 4 advantageously comprise plastic. In accordance with the statements made in the foregoing, each of the one-piece sleeves 4 comprises the two sleeve parts, with the integrated retaining element 6, that have been joined in a material fit in particular by welding. The preferably four viewing windows 58 embodied as through-holes are easy to see here. The statements made in the foregoing apply as appropriate.

Fig. 9 illustrates another exemplary embodiment of a fitting embodied as a so-called wall disk with a plug-in coupling containing a sleeve 4. The base body 2 is a component of the fitting body 98, which is embodied for attaching another tube, in particular a metal tube, via a connecting device that is not shown here and that is provided in the interior of the fitting body 98, preferably a threaded connector. The fitting body 98 and the base body 2 of the plug-in coupling, embodied integrally therewith, preferably comprise metal, in particular brass, while the inventive plug-in coupling with the sleeve 4 that preferably comprises plastic is fundamentally in agreement with the exemplary embodiments explained in the foregoing. The fitting base body 98 contains a flange-like fastening body 100 for connecting to a wall, a mounting system, or the like.

REFERENCE CHARACTERS

2	Base body
4	Sleeve

	6	Retaining element
	8	Sealing element
	10	Annular gap
	12	Retaining claw
5	14	Anterior free sleeve end
	16	Distance
	18	Free end of 2
	20	Locking means/bar of 2
	22	Locking means/annular groove of 4
10	24	Longitudinal axis
	26	Interior surface of 30
	28	Interior surface of 32
	30	First anterior sleeve part
	32	Second posterior sleeve part
15	34	Support body/conical annular shoulder
	38	Claw edge
	40	Connecting part of 30
	42	Connecting part of 32
	44	Closed annular part of 6
20	46	Arrow

	48	Interior surface part of 32
	50	Exterior positioning surface of 34
	52	Angle of 50
	54	Angle of 48
5	56	Free space
	58	Through-hole/viewing window
	60	Tube end
	62	Metal intermediate layer
	64	Arrow
10	66	Angling-off
	68	Recess
	70	Through-hole in 6
	72	Expansion of 12
	74	Center part of 12
15	76	Exterior edge of 12
	78	Radius of 76
	80	Mean radius of 38
	82	Locking element of 30
	84	Locking element of 32
20	86	Annular groove in 2

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- 88 Annular body in 86
- 90 Recess in 8
- 92 Anterior section of 8
- 94 Adhesive connection
- 5 96 Posterior section of 8
- 98 Fitting base body
- 100 Fastening body